

Если выделить двумя бесконечно близкими поперечными сечениями элемент стержня длиной  $\delta x$ , то силы инерции, соответствующие вращению этого элемента, дадут нам моментъ равный

$$- \frac{J \gamma}{g} \frac{\partial^2 y}{\partial x \partial t^2} \delta x \dots \dots \dots (c)$$

При составлении дифференциальнаго ур-ня для изогнутой оси стержня нужно принять во вниманіе не только непрерывно распределенную нагрузку, соответствующую силамъ инерции поступательнаго движенія, но и непрерывно распределенные моменты, опредѣляемые  $\phi$ -ой (c). Въ такомъ случаѣ вліяніе изгибающаго момента вдоль оси стержня представится такъ

$$\frac{\partial M}{\partial x} = Q - \frac{J \gamma}{g} \frac{\partial^2 y}{\partial x \partial t^2}$$

Здѣсь  $Q$  поперечная сила, получающаяся отъ силы инерции поступательнаго движенія элемента стержня.

Вставляя найденное значеніе производной  $\frac{\partial M}{\partial x}$  въ дифференціальное ур-іе

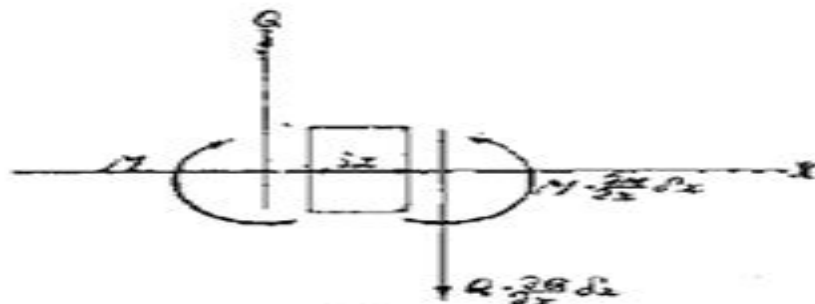
$$EJ \frac{\partial^2 y}{\partial x^2} = - \frac{\partial M}{\partial x},$$

получаемъ

$$EJ \frac{\partial^4 y}{\partial x^4} = - \frac{\gamma F}{g} \frac{\partial^2 y}{\partial t^2} + \frac{J \gamma}{g} \frac{\partial^4 y}{\partial x^2 \partial t^2} \dots \dots \dots (177)$$

Второй членъ въ правой части ур-ія представляетъ собой поправку, обусловленную поворотомъ поперечныхъ сеченій при колебаніяхъ.

Болѣе точное ур-іе мы получимъ, если примемъ во вниманіе не только вліяніе силъ инерции, соответствующихъ повороту сѣченій, но также и вліяніе касательныхъ напряженій на прогибъ стержня<sup>1)</sup>. Выделимъ изъ стержня двумя бесконечно близкими сѣченіями элементъ длиной  $\delta x$  и составимъ для него ур-іе движенія. Положеніе выделеннаго элемента опредѣлится по размѣщенію  $y$  его центра тяжести и угломъ поворота  $\phi$  относительно нейтральной линіи<sup>2)</sup>. Такъ какъ прогибъ



Черт. 76.

стержня обусловленъ не только относительно поворотомъ поперечныхъ сѣченій, но также и движомъ ихъ одно относительно другого, то уголъ наклона кас-

<sup>1)</sup> Вліяніе касательныхъ напряженій на періодъ поперечныхъ колебаній стержня обыкновенно не принимается въ расчетъ, а между тѣмъ соответствующая ему поправка по своей величинѣ больше той, которая получается отъ поперечнаго сѣченія.

<sup>2)</sup> При составленіи ур-ія мы рассматриваемъ элементъ, какъ абсолютно твердое тѣло и пренебрегаемъ искривленіемъ поперечныхъ сѣченій, поэтому наша поправка лишь приближенно оцѣниваетъ вліяніе касательныхъ силъ. Нами совместно съ проф. П. Эренфестомъ получено также точное рѣшеніе задачи для колеса прямоугольнаго поперечнаго сѣченія. Это рѣшеніе, какъ и цитированное нами выше (стр. 184) рѣшеніе Рессламмгофа для круглаго цилиндра, показываютъ, что приведенное здѣсь приближенное рѣшеніе съ достаточной точностью учитываетъ вліяніе касательныхъ силъ.

# Elasticity Timoshenko Solution Manual

**LL Leslie**



## **Elasticity Timoshenko Solution Manual:**

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**Improved Numerical Methods for Solutions of a Beam's Axial, Torsion and Flexure Problems** Zvi Friedman, 1997 **A Project-Based Introduction to Computational Statics** Andreas Öchsner, 2020-11-13 This book uses a novel concept to teach the finite element method applying it to solid mechanics This major conceptual shift takes away lengthy theoretical derivations in the face to face interactions with students and focuses on the summary of key equations and concepts and to practice these on well chosen example problems For this new 2nd edition many examples and design modifications have been added so that the learning by doing features of this book make it easier to understand the concepts and put them into practice The theoretical derivations are provided as additional reading and students must study and review the derivations in a self study approach The book provides the theoretical foundations to solve a comprehensive design project in tensile testing A classical clip on extensometer serves as the demonstrator on which to apply the provided concepts The major goal is to derive the calibration curve based on different approaches i e analytical mechanics and based on the finite element method and to consider further design questions such as technical drawings manufacturing and cost assessment Working with two concepts i e analytical and computational mechanics strengthens the vertical integration of knowledge and allows the student to compare and understand the different concepts as well as highlighting the essential need for benchmarking any numerical result

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*Journal of the Aeronautical Sciences* ,1940      **ASME Technical Papers** ,1980      **The Finite Element Method in  
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